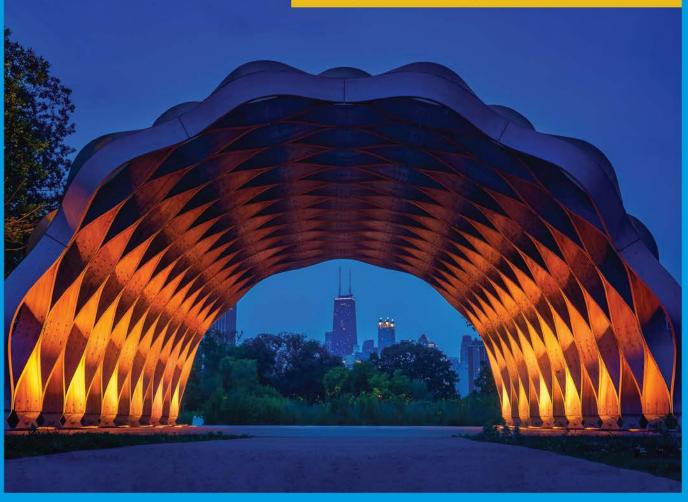


Enhanced 9th Edition

MECHANICS OF MATERIALS

Barry J. Goodno | James M. Gere



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CONVERSIONS BETWEEN U.S. CUSTOMARY UNITS AND SI UNITS

	Times conversion factor				
U.S. Customary unit		Accurate Practical		Equals SI unit	
Acceleration (linear) foot per second squared inch per second squared	ft/s ² in./s ²	0.3048* 0.0254*	0.305 0.0254	meter per second squared meter per second squared	m/s ² m/s ²
Area square foot square inch	ft ² in. ²	0.09290304* 645.16*	0.0929 645	square meter square millimeter	m ² mm ²
Density (mass) slug per cubic foot	slug/ft ³	515.379	515	kilogram per cubic meter	kg/m ³
Density (weight) pound per cubic foot pound per cubic inch	lb/ft ³ lb/in. ³	157.087 271.447	157 271	newton per cubic meter kilonewton per cubic meter	N/m ³ kN/m ³
Energy; work foot-pound inch-pound kilowatt-hour British thermal unit	ft-lb inlb kWh Btu	1.35582 0.112985 3.6* 1055.06	1.36 0.113 3.6 1055	joule (N·m) joule megajoule joule	J J MJ J
Force pound kip (1000 pounds)	lb k	4.44822 4.44822	4.45 4.45	newton (kg·m/s ²) kilonewton	N kN
Force per unit length pound per foot pound per inch kip per foot kip per inch	lb/ft lb/in. k/ft k/in.	14.5939 175.127 14.5939 175.127	14.6 175 14.6 175	newton per meter newton per meter kilonewton per meter kilonewton per meter	N/m N/m kN/m kN/m
Length foot inch mile	ft in. mi	0.3048* 25.4* 1.609344*	0.305 25.4 1.61	meter millimeter kilometer	m mm km
Mass slug	lb-s ² /ft	14.5939	14.6	kilogram	kg
Moment of a force; torque pound-foot pound-inch kip-foot kip-inch	lb-ft lb-in. k-ft k-in.	1.35582 0.112985 1.35582 0.112985	1.36 0.113 1.36 0.113	newton meter newton meter kilonewton meter kilonewton meter	N∙m N∙m kN∙m kN∙m

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U.C. C		Times convers	ion factor	E-male CI		
U.S. Customary unit		Accurate	Practical	Equals SI unit	i unit	
Moment of inertia (area) inch to fourth power	in. ⁴	416,231	416,000	millimeter to fourth	mm ⁴	
inch to fourth power	in. ⁴	0.416231×10^{-6}	0.416×10^{-6}	meter to fourth power	m^4	
Moment of inertia (mass) slug foot squared	slug-ft ²	1.35582	1.36	kilogram meter squared	kg·m ²	
Power						
foot-pound per second	ft-lb/s	1.35582	1.36	watt (J/s or N·m/s)	W	
foot-pound per minute	ft-lb/min	0.0225970	0.0226	watt	W	
horsepower (550 ft-lb/s)	hp	745.701	746	watt	W	
Pressure; stress						
pound per square foot	psf	47.8803	47.9	pascal (N/m ²)	Pa	
pound per square inch	psi	6894.76	6890	pascal	Pa	
kip per square foot	ksf	47.8803	47.9	kilopascal	kPa	
kip per square inch	ksi	6.89476	6.89	megapascal	MPa	
Section modulus						
inch to third power	in. ³	16,387.1	16,400	millimeter to third power	mm ³	
inch to third power	in. ³	16.3871×10^{-6}	16.4×10^{-6}	meter to third power	m ³	
Velocity (linear)						
foot per second	ft/s	0.3048*	0.305	meter per second	m/s	
inch per second	in./s	0.0254*	0.0254	meter per second	m/s	
mile per hour	mph	0.44704*	0.447	meter per second	m/s	
mile per hour	mph	1.609344*	1.61	kilometer per hour	km/h	
Volume						
cubic foot	ft^3	0.0283168	0.0283	cubic meter	m ³	
cubic inch	in. ³	16.3871×10^{-6}	16.4×10^{-6}	cubic meter	m ³	
cubic inch	in. ³	16.3871	16.4	cubic centimeter (cc)	cm ³	
gallon (231 in. ³)	gal.	3.78541	3.79	liter	L	
gallon (231 in. ³)	gal.	0.00378541	0.00379	cubic meter	m ³	

CONVERSIONS BETWEEN U.S. CUSTOMARY UNITS AND SI UNITS (Continued)

*An asterisk denotes an exact conversion factor

Note: To convert from SI units to USCS units, divide by the conversion factor

Temperature Conversion Formulas

$$T(^{\circ}C) = \frac{5}{9}[T(^{\circ}F) - 32] = T(K) - 273.15$$
$$T(K) = \frac{5}{9}[T(^{\circ}F) - 32] + 273.15 = T(^{\circ}C) + 273.15$$
$$T(^{\circ}F) = \frac{9}{5}T(^{\circ}C) + 32 = \frac{9}{5}T(K) - 459.67$$

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Mechanics of Materials

Enhanced Ninth Edition

Barry J. Goodno

Georgia Institute of Technology

James M. Gere Professor Emeritus, Stanford University



Australia • Brazil • Mexico • Singapore • United Kingdom • United States

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*A star attached to a section number indicates a specialized and/or advanced topic.

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ABOUT THE AUTHORS

Barry J. Goodno

Barry John Goodno is Professor of Civil and Environmental Engineering at Georgia Institute of Technology. He joined the Georgia Tech faculty in 1974. He was an Evans Scholar and received a B.S. in Civil Engineering from the University of Wisconsin, Madison, Wisconsin, in 1970. He received M.S. and Ph.D. degrees in Structural Engineering from Stanford University, Stanford, California, in 1971 and 1975, respectively. He holds a professional engineering license (PE) in Georgia, is a Distinguished Member of ASCE and an Inaugural Fellow of SEI, and has held numerous leadership positions within ASCE. He is a past president of the ASCE Structural Engineering Institute (SEI) Board of Governors and is also a member of the Engineering Mechanics Institute (EMI) of ASCE. He is past-chair of the ASCE-SEI Technical Activities Division (TAD) Executive Committee, and past-chair of the ASCE-SEI Awards Committee. In 2002, Dr. Goodno received the SEI Dennis L. Tewksbury Award for outstanding service to ASCE-SEI. He received the departmental award for *Leadership in Use of Technology* in 2013 for his pioneering use of lecture capture technologies in undergraduate statics and mechanics of materials courses at Georgia Tech. He is a member of the Earthquake Engineering Research Institute (EERI) and has held several leadership positions within the NSF-funded Mid-America Earthquake Center (MAE), directing the MAE Memphis Test Bed Project. Dr. Goodno has carried out research, taught graduate courses and published extensively in the areas of earthquake engineering and structural dynamics during his tenure at Georgia Tech.

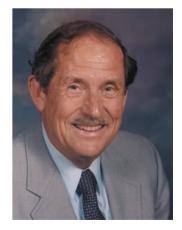
Dr. Goodno is an active cyclist, retired soccer coach and referee, and a retired marathon runner. Like co-author and mentor James Gere, he has completed numerous marathons including qualifying for and running the Boston Marathon in 1987.



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James M. Gere

James M. Gere (1925-2008) earned his undergraduate and master's degree in Civil Engineering from the Rensselaer Polytechnic Institute in 1949 and 1951, respectively. He worked as an instructor and later as a Research Associate for Rensselaer. He was awarded one of the first NSF Fellowships, and chose to study at Stanford. He received his Ph.D. in 1954 and was offered a faculty position in Civil Engineering, beginning a 34-year career of engaging his students in challenging topics in mechanics, and structural and earthquake engineering. He served as Department Chair and Associate Dean of Engineering and in 1974 co-founded the John A. Blume Earthquake Engineering Center at Stanford. In 1980, Jim Gere also became the founding head of the Stanford Committee on Earthquake Preparedness. That same year, he was invited as one of the first foreigners to study the earthquake-devastated city of Tangshan, China. Jim retired from Stanford in 1988 but continued to be an active and most valuable member of the Stanford community.



Courtesy of James and Janice Gere Family Trust

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Jim Gere was known for his outgoing manner, his cheerful personality and wonderful smile, his athleticism, and his skill as an educator in Civil Engineering. He authored nine textbooks on various engineering subjects starting in 1972 with *Mechanics of Materials*, a text that was inspired by his teacher and mentor Stephan P. Timoshenko. His other well-known textbooks, used in engineering courses around the world, include: *Theory of Elastic Stability*, co-authored with S. Timoshenko; *Matrix Analysis of Framed Structures* and *Matrix Algebra for Engineers*, both co-authored with W. Weaver; *Moment Distribution; Earthquake Tables: Structural and Construction Design Manual*, co-authored with H. Krawinkler; and *Terra Non Firma: Understanding and Preparing for Earthquakes*, co-authored with H. Shah.

In 1986 he hiked to the base camp of Mount Everest, saving the life of a companion on the trip. James was an active runner and completed the Boston Marathon at age 48, in a time of 3:13. James Gere will be long remembered by all who knew him as a considerate and loving man whose upbeat good humor made aspects of daily life or work easier to bear.

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PREFACE

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Mechanics of Materials is a basic engineering subject that, along with statics, must be understood by anyone concerned with the strength and physical performance of structures, whether those structures are man-made or natural. At the college level, statics is usually taught during the sophomore or junior year and is a prerequisite for the follow-on course in Mechanics of Materials. Both courses are required for most students majoring in mechanical, structural, civil, biomedical, petroleum, nuclear, aeronautical, and aerospace engineering. In addition, many students from such diverse fields as materials science, industrial engineering, architecture, and agricultural engineering also find it useful to study mechanics of materials.

Mechanics of Materials

In many university engineering programs today, both statics and mechanics of materials are taught in large sections of students from the many engineering disciplines. Instructors for the various parallel sections must cover the same material, and all of the major topics must be presented so that students are well prepared for the more advanced courses required by their specific degree programs. An essential prerequisite for success in a first course in mechanics of materials is a strong foundation in statics, which includes not only understanding fundamental concepts but also proficiency in applying the laws of static equilibrium to solutions of both two- and three-dimensional problems. This ninth edition begins with an updated section on statics in which the laws of equilibrium and an expanded list of boundary (or support) conditions are reviewed, as well as types of applied forces and internal stress resultants, all based upon and derived from a properly drawn free-body diagram. Numerous examples and endof-chapter problems are included to help students review the analysis of plane and space trusses, shafts in torsion, beams and plane and space frames, and to reinforce basic concepts learned in the prerequisite course.

Many instructors like to present the basic theory of say, beam bending, and then use real world examples to motivate student interest in the subject of beam flexure, beam design, etc. In many cases, structures on campus offer easy access to beams, frames, and bolted connections that can be dissected in lecture or in homework problems, to find reactions at supports, forces and moments in members and stresses in connections. In addition, study of causes of failures in structures and components also offers the opportunity for students to begin the process of learning from actual designs and past engineering mistakes. A number of the new example problems and also the new and revised end-of-chapter problems in this ninth edition are based upon actual components or structures and are accompanied by photographs so that the student can see the real world problem alongside the simplified mechanics model and free-body diagrams used in its analysis.

An increasing number of universities are using rich media lecture (and/ or classroom) capture software (such as Panopto and Tegrity) in their large undergraduate courses in mathematics, physics, and engineering. The *many new photos and enhanced graphics* in the ninth edition are designed to support this enhanced lecture mode.

Key Features

The main topics covered in this book are the analysis and design of structural members subjected to tension, compression, torsion, and bending, including the fundamental concepts mentioned above. Other important topics are the transformations of stress and strain, combined loadings and combined stress, deflections of beams, and stability of columns. Some additional specialized topics include the following: stress concentrations, dynamic and impact loadings, non-prismatic members, shear centers, bending of beams of two materials (or composite beams), bending of unsymmetric beams, maximum stresses in beams, energy based approaches for computing deflections of beams, and statically indeterminate beams.

Each chapter begins with the *Chapter Objectives* and a *Chapter Outline* highlighting the major topics covered in that chapter and closes with a *Chapter Summary and Review* in which the key points as well as major mathematical formulas in the chapter are listed for quick review. Each chapter also opens with a photograph of a component or structure that illustrates the key concepts discussed in the chapter.

New Features

Some of the notable features of this ninth edition, which have been added as new or updated material to meet the needs of a modern course in mechanics of materials, are:

- **Problem-Solving Approach**—All examples in the text are presented in a new Four-Step Problem-Solving Approach which is patterned after that presented by R. Serway and J. Jewett in *Principles of Physics*, 5e, Cengage Learning, 2013. This new structured format helps students refine their problem-solving skills and improve their understanding of the main concepts illustrated in the example.
- **Statics Review**—The *Statics Review* section has been enhanced in Chapter 1. Section 1.2 includes four new example problems which illustrate calculation of support reactions and internal stress resultants for truss, beam, circular shaft and plane frame structures. Thirty-four end-of-chapter problems on statics provide students with two- and three-dimensional structures to be used as practice, review, and homework assignment problems of varying difficulty.
- Expanded Chapter Objectives and Chapter Summary and Review sections— The *Chapter Objectives* are listed at the beginning of each chapter and the *Chapter Summary* section has been expanded to include *key equations* and *figures* presented in each chapter. These summary sections serve as a convenient review for students of key topics and equations presented in each chapter.
- **Continued emphasis on underlying fundamental concepts** such as equilibrium, constitutive, and strain-displacement/ compatibility equations in problem solutions. Example problem and end-of-chapter problem solutions have been updated to emphasize an orderly process of explicitly writing out the equilibrium, constitutive and strain-displacement/ compatibility equations before attempting a solution.

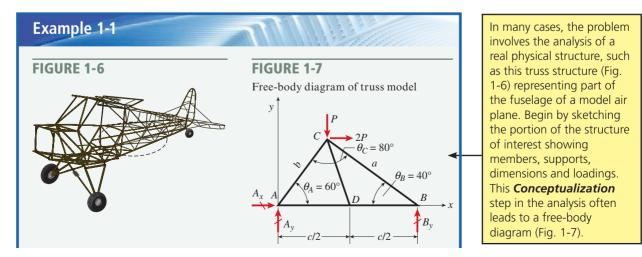
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- Expanded topic coverage—The following topics have been updated or have received expanded coverage: stress concentrations in axially loads bars (Sec. 2.10); torsion of noncircular shafts (Sec. 3.10); stress concentrations in bending (Sec. 5.13); transformed section analysis for composite beams (Sec. 6.3); generalized flexure formula for unsymmetric beams (Sec. 6.5); and updated code provisions for buckling of steel, aluminum and timber columns (Sec. 11.9).
- Many new example and end-of-chapter problems—More than forty new example problems have been added to the ninth edition. In addition, there are more than 400 new and revised end-of-chapter problems out of the 1440 problems presented in the ninth edition text. The end-of-chapter problems are now grouped as **Introductory** or **Representative** and are arranged in order of increasing difficulty.
- Centroids and Moments of Inertia review has been moved to Appendix D to free up space for more examples and problems in earlier chapters.

Importance of Example Problems

• Examples are presented throughout the book to illustrate the theoretical concepts and show how those concepts may be used in practical situations. All examples are presented in the **Four-Step Problem-Solving Approach** format so that the basic concepts as well as the key steps in setting up and solving each problem are clearly understood. New photographs have been added showing actual engineering structures or components to reinforce the tie between theory and application. Each example begins with a clear statement of the problem and then presents a simplified analytical model and the associated free-body diagrams to aid students in understanding and applying the relevant theory in engineering analysis of the system. In most cases, the examples are worked out in symbolic terms so as to better illustrate the ideas, and then numeric values of key parameters are substituted in the final part of the analysis step. In selected examples throughout the text, graphical display of results (e.g., stresses in beams) has been added to enhance the student's understanding of the problem results.



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The next step is to simplify the problem, list known data and identify all unknowns, and make necessary assumptions to create a suitable model for analysis. This is the **Categorize** step.

Write the governing equations, then use appropriate mathematical and computational techniques to solve the equations and obtain results, either in the form of mathematical formulas or numerical values. The **Analysis** step leads to support reaction and member forces in the truss.

List the major steps in your analysis procedure so that it is easy to review or check at a later time.

Solution:

The solution involves the following steps:

- 1. *Conceptualize* [*hypothesize, sketch*]: First sketch a free-body diagram of the entire truss model (Figure 1-7). Only known applied forces at *C* and unknown reaction forces at *A* and *B* are shown and then used in an equilibrium analysis to find the reactions.
- 2. Categorize [simplify, classify]: Overall equilibrium requires that the force components in x and y directions and the moment about the z axis must sum to zero; this leads to reaction force components A_x , A_y , and B_y . The truss is statically determinate (unknowns: m + r = 5 + 3 = 8, knowns: 2j = 8) so all member forces can be obtained using the method of joints. . . .
- 3. Analyze [evaluate; select relevant equations, carry out mathematical *solution*]: First find the lengths of members AC and BC, which are needed to compute distances to lines of action of forces.

Law of sines to find member lengths *a* **and** *b***:** Use known angles θ_A , θ_B , and θ_C and c = 10 ft to find lengths *a* and *b*:

$$b = c \frac{\sin(\theta_B)}{\sin(\theta_C)} = (10 \text{ ft}) \frac{\sin(40^\circ)}{\sin(80^\circ)} = 6.527 \text{ ft},$$
$$a = c \frac{\sin(\theta_A)}{\sin(\theta_C)} = (10 \text{ ft}) \frac{\sin(60^\circ)}{\sin(80^\circ)} = 8.794 \text{ ft}$$

Check that computed lengths *a* and *b* give length *c* by using the law of cosines:

$$c = \sqrt{(6.527 \text{ ft})^2 + (8.794 \text{ ft})^2 - 2(6.527 \text{ ft})(8.794 \text{ ft})\cos(80^\circ)} = 10 \text{ ft}$$

4. *Finalize* [conclude; examine answer—does it make sense? Are units correct? How does it compare to similar problem solutions?]: There are 2 j = 8 equilibrium equations for the simple plane truss considered above and, using the *method of joints*, these are obtained by applying $\Sigma F_x = 0$ and $\Sigma F_y = 0$ at each joint in succession. A computer solution of these simultaneous equations leads to the three reaction forces and five member forces. The *method of sections* is an efficient way to find selected member forces.

List the major steps in the *Finalize* step, review the solution to make sure that it is presented in a clear fashion so that it can be easily reviewed and checked by others. Are the expressions and numerical values obtained reasonable? Do they agree with your initial expectations?

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Problems

In all mechanics courses, solving problems is an important part of the learning process. This textbook offers more than 1440 problems, many with multiple parts, for homework assignments and classroom discussions. The problems are placed at the end of each chapter so that they are easy to find and don't break up the presentation of the main subject matter. Also, problems are generally arranged in order of increasing difficulty, thus alerting students to the time necessary for solution. Answers to all problems are listed near the back of the book.

Considerable effort has been spent in checking and proofreading the text so as to eliminate errors. If you happen to find one, no matter how trivial, please notify me by e-mail (*bgoodno@ce.gatech.edu*). We will correct any errors in the next printing of the book.

Units

Both the International System of Units (SI) and the U.S. Customary System (USCS) are used in the examples and problems. Discussions of both systems and a table of conversion factors are given in Appendix A. For problems involving numerical solutions, odd-numbered problems are in USCS units and evennumbered problems are in SI units. This convention makes it easy to know in advance which system of units is being used in any particular problem. In addition, tables containing properties of structural-steel shapes in both USCS and SI units may be found in Appendix F so that solution of beam analysis and design examples and end-of-chapter problems can be carried out in either USCS or SI units.

Supplements

Instructor Resources

An **Instructor's Solutions Manual** is available online on the Instructor's Resource Center for this book, and includes solutions to all problems from this edition with Mathcad solutions available for some problems. The Manual includes rotated stress elements for problems as well as an increased number of free body diagrams. The Solutions Manual is accessible to instructors on http://login.cengage.com. The Instructor Resource Center also contains a full set of Lecture Note PowerPoints.

Student Resources

FE Exam Review Problems has been updated and now appears online. This supplement contains 106 FE-type review problems and solutions, which cover all of the major topics presented in the text and are representative of those likely to appear on an FE exam. Each of the problems is presented in the FE Exam format and is intended to serve as a useful guide to the student in preparing for this important examination.

Many students take the *Fundamentals of Engineering Examination* upon graduation, the first step on their path to registration as a Professional Engineer. Most of these problems are in SI units which is the system of units used

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on the FE Exam itself, and require use of an engineering calculator to carry out the solution. The student must select from four available answers, only one of which is the correct answer. Go to http://login.cengage.com to find the FE Exam Review Problems and the resources below, which are available on the student website for this book:

- Answers to the FE Exam Review Problems
- Detailed Solutions for Each Problem

S.P. Timoshenko (1878–1972) and J.M. Gere (1925–2008)

Many readers of this book will recognize the name of Stephen P. Timoshenko probably the most famous name in the field of applied mechanics. A brief biography of Timoshenko appears in the first reference in the References and Historical Notes section. Timoshenko is generally recognized as the world's most outstanding pioneer in applied mechanics. He contributed many new ideas and concepts and became famous for both his scholarship and his teaching. Through his numerous textbooks he made a profound change in the teaching of mechanics not only in this country but wherever mechanics is taught. Timoshenko was both teacher and mentor to James Gere and provided the motivation for the first edition of this text, authored by James M. Gere and published in 1972. The second and each subsequent edition of this book were written by James Gere over the course of his long and distinguished tenure as author, educator, and researcher at Stanford University. James Gere started as a doctoral student at Stanford in 1952 and retired from Stanford as a professor in 1988 having authored this and eight other well-known and respected text books on mechanics, and structural and earthquake engineering. He remained active at Stanford as Professor Emeritus until his death in January of 2008.

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I wish to acknowledge and thank the Global Engineering team at Cengage for their dedication to this new book:

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They have skillfully guided every aspect of this text's development and production to successful completion.

I am deeply appreciative of the patience and encouragement provided by my family, especially my wife, Lana, throughout this project.

Finally, I am very pleased to continue this endeavor begun so many years ago by my mentor and friend, Jim Gere. This ninth edition text has now reached its 45th year of publication. I am committed to its continued excellence and welcome all comments and suggestions. Please feel free to provide me with your critical input at *bgoodno@ce.gatech.edu*.

Barry J. Goodno Atlanta, Georgia

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DIGITAL RESOURCES

CENGAGE | WEBASSIGN

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Reader at the precise moment of learning. A "Watch It" button allows a short video to play. These videos help students understand and review the problem they need to complete, enabling support at the precise moment of learning.

D2 peters (Preview Answer) GOMechilar9 5 13 005
A rectangular beam with notiches and a hole (see figure) has dimensions $h + 7.7$ in, $h_1 = 7$ in, and thickness $b = 0.73$ in. (perpendicular to the plane of the figure). $M = \frac{2R}{h_1} + \frac{2R}{h_2} + \frac{2R}{h_3} + \frac{2R}{h_4} + \frac{2R}{h_5} + \frac{2R}{h$
[©]
The beam is subjected to a bending moment M = 125 kip-in., and the maximum allowable bending stress in the material (steel) is σ_{max} = 42,000 psl.
(a) What is the smallest radius Amin (in inches) that should be used in the notches? (Use any necessary data found in this figure.)
125 🗶 0.77 in.
(b) What is the diameter d_{max} (in inches) of the largest hole that should be drilled at the midheight of the beam? (Assume that the ratio $\frac{d}{h} > \frac{1}{2}$.)
25 🗙 🐱 5.65 (n.
Need Help? Reven

Four-Step Active Examples

Throughout this title, selected examples guide students through a four-step process needed to master concepts. Worked-out solutions are included for these examples.

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2.4 Statically Indeterminate Structures



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plane of bendi	ng	Next Car
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SYMBOLS

- A area
- $A_{\rm f}, A_{\rm w}$ area of flange; area of web
- a, b, c dimensions, distances
 - C centroid, compressive force, constant of integration
 - c distance from neutral axis to outer surface of a beam
 - D diameter
 - d diameter, dimension, distance
 - E modulus of elasticity
- E_r, E_t reduced modulus of elasticity; tangent modulus of elasticity
 - *e* eccentricity, dimension, distance, unit volume change (dilatation)
 - F force
 - f shear flow, shape factor for plastic bending, flexibility, frequency (Hz)
 - f_T torsional flexibility of a bar
 - G modulus of elasticity in shear
 - g acceleration of gravity
 - H height, distance, horizontal force or reaction, horsepower
 - *h* height, dimensions
 - I moment of inertia (or second moment) of a plane area
- I_x, I_y, I_z moments of inertia with respect to x, y, and z axes
 - I_{x1} , I_{y1} moments of inertia with respect to x_1 and y_1 axes (rotated axes)
 - I_{xy} product of inertia with respect to xy axes
 - I_{x1y1} product of inertia with respect to x_1y_1 axes (rotated axes)
 - I_p polar moment of inertia
 - I_1, I_2 principal moments of inertia
 - J torsion constant
 - K stress-concentration factor, bulk modulus of elasticity, effective length factor for a column
 - k spring constant, stiffness, symbol for $\sqrt{P/EI}$
 - k_T torsional stiffness of a bar
 - L length, distance
 - L_E effective length of a column
 - ln, log natural logarithm (base e); common logarithm (base 10)
 - *M* bending moment, couple, mass
- M_P, M_Y plastic moment for a beam; yield moment for a beam
 - *m* moment per unit length, mass per unit length
 - N axial force

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ΧХ

- *n* factor of safety, integer, revolutions per minute (rpm)
- O origin of coordinates
- O' center of curvature
- *P* force, concentrated load, power
- P_{allow} allowable load (or working load)
 - $P_{\rm cr}$ critical load for a column
 - P_p plastic load for a structure
- P_r, P_t reduced-modulus load for a column; tangent-modulus load for a column
 - P_{Y} yield load for a structure
 - *p* pressure (force per unit area)
 - Q force, concentrated load, first moment of a plane area
 - q intensity of distributed load (force per unit distance)
 - R reaction, radius
 - *r* radius, radius of gyration ($r = \sqrt{I/A}$)
 - S section modulus of the cross section of a beam, shear center
 - s distance, distance along a curve
 - T tensile force, twisting couple or torque, temperature
- T_P, T_Y plastic torque; yield torque
 - t thickness, time, intensity of torque (torque per unit distance)
 - $t_{\rm f}, t_{\rm w}$ thickness of flange; thickness of web
 - U strain energy
 - *u* strain-energy density (strain energy per unit volume)
- u_r, u_t modulus of resistance; modulus of toughness
 - V shear force, volume, vertical force or reaction
 - v deflection of a beam, velocity
- v', v'', etc. $dv/dx, d^2v/dx^2$, etc.
 - W force, weight, work
 - w load per unit of area (force per unit area)
 - x, y, z rectangular axes (origin at point O)
- x_c, y_c, z_c rectangular axes (origin at centroid C)
 - $\overline{x}, \overline{y}, \overline{z}$ coordinates of centroid
 - Z plastic modulus of the cross section of a beam
 - α angle, coefficient of thermal expansion, nondimensional ratio
 - β angle, nondimensional ratio, spring constant, stiffness
 - β_R rotational stiffness of a spring
 - γ shear strain, weight density (weight per unit volume)

 $\gamma_{xy}, \gamma_{yz}, \gamma_{zx}$ shear strains in *xy*, *yz*, and *zx* planes

- γ_{x1y1} shear strain with respect to x_1y_1 axes (rotated axes)
 - γ_{θ} shear strain for inclined axes
 - δ deflection of a beam, displacement, elongation of a bar or spring

- ΔT temperature differential
- δ_P, δ_V plastic displacement; yield displacement
 - ε normal strain
- $\varepsilon_x, \varepsilon_y, \varepsilon_z$ normal strains in x, y, and z directions
 - $\varepsilon_{x1}, \varepsilon_{y1}$ normal strains in x_1 and y_1 directions (rotated axes)
 - ε_{θ} normal strain for inclined axes
- $\varepsilon_1, \varepsilon_2, \varepsilon_3$ principal normal strains
 - ε' lateral strain in uniaxial stress
 - ε_T thermal strain
 - ε_Y yield strain
 - θ angle, angle of rotation of beam axis, rate of twist of a bar in torsion (angle of twist per unit length)
 - θ_n angle to a principal plane or to a principal axis
 - θ_s angle to a plane of maximum shear stress
 - κ curvature ($\kappa = 1/\rho$)
 - λ distance, curvature shortening
 - ν Poisson's ratio
 - ρ radius, radius of curvature ($\rho = 1/\kappa$), radial distance in polar coordinates, mass density (mass per unit volume)
 - σ normal stress
- $\sigma_x, \sigma_y, \sigma_z$ normal stresses on planes perpendicular to x, y, and z axes
 - σ_{x1}, σ_{y1} normal stresses on planes perpendicular to x_1y_1 axes (rotated axes)
 - σ_{θ} normal stress on an inclined plane
- $\sigma_1, \sigma_2, \sigma_3$ principal normal stresses
 - σ_{allow} allowable stress (or working stress)
 - $\sigma_{\rm cr}$ critical stress for a column ($\sigma_{\rm cr} = P_{\rm cr}/A$)
 - $\sigma_{\rm pl}$ proportional-limit stress
 - σ_r residual stress
 - σ_T thermal stress
 - σ_{II}, σ_{Y} ultimate stress; yield stress
 - au shear stress

 $\tau_{xy}, \tau_{yz}, \tau_{zx}$

- shear stresses on planes perpendicular to the *x*, *y*, and *z* axes and acting parallel to the *y*, *z*, and *x* axes
- τ_{x1y1} shear stress on a plane perpendicular to the x_1 axis and acting parallel to the y_1 axis (rotated axes)
 - τ_{θ} shear stress on an inclined plane
- $\tau_{\rm allow}$ allowable stress (or working stress) in shear
- τ_U, τ_Y ultimate stress in shear; yield stress in shear
 - ϕ angle, angle of twist of a bar in torsion
 - ψ angle, angle of rotation
 - ω angular velocity, angular frequency ($\omega = 2\pi f$)

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